20124551-2

Team 510: Climatic Camera

Design Review VI

Nash Bonaventura Diego Gonzalez Bryce Shumaker

Department of Mechanical Engineering





Team Introductions



Diego Gonzalez Design Engineer



Nash Bonaventura Simulation Engineer



Bryce Shumaker Project Manager





Stakeholders





Engineering Mentor Kourosh Shoele, Ph.D. Assistant Professor FAMU-FSU College of Engineering <u>Sponsor</u> Vinayak Hegde, *Reliability Engineering Manager*

Danfoss Turbocor Compressors, Inc.

Diego Gonzalez





Project Objective

The objective of the project is to design a product that will maintain operation of a recording device at extreme temperatures (-40 to 160 °C)

(-40 to 320 °F)







Background

- Air compressor manufacturer
- Components tested by reliability engineering department
- Components are tested using cyclic temperature tests
- Test Temperature range (-40 to 160 °C)
- Cameras operates between 0 and 45 °C





are

THERMOTRO



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Current Problems

- Physical presence is necessary to monitor
- Window gets foggy and obstructs view
- Reflection from window
- Outside Visuals
 - Fixed viewing distance
 - Low reachability





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Customer Needs

The device provides live continuous monitoring The device is isolated from the testing environment The device can be adjusted to different orientations The device has computer connection capabilities The device has failure detection capabilities







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Engineering





Available Resources

- Compressed Air temperature regulation
- Laptop power supply, software interface, data storage
- Chamber Port connection with auxiliary systems
- Racks mounting
- Machine Shop





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FAMILESI





Selected Concept

Compressed air, USB Borescope Camera



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Selected Design





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Overall System



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Design Functions: Isolate







Design Functions: Support

Device



Stainless Steel Plate Welded to Hose Clamps



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Gooseneck

Magnetic

Base





Design Functions: Alarm System Speaker Computer Temperature Camera Connection Sensor Housing Power Arduino Cable **Bryce Shumaker**





Current Design Problems

Borescope camera LED light reflection in Glass



Bryce Shumaker

OPTICLEAR

GLASS





Budgeting

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Items Received Items Requested

	Item #	Item Description	Quanti	Price
	1	Borescope Camera	1	49.99
1	2	Glass Housing	1	22.52
	3	Housing Body	1	12.22
2	4	Clamp for Housing	1	\$11.39
-	5	Clamp set	1	\$11.90
100	6	Goose neck	1	\$9.95
	7	Magnet base	1	\$16.99
	8	Desicant Air Dryer	1	\$129.99
	9	Shut-Off Brass Valve	1	\$8.98
	10	Brass Reducer	1	\$3.79
	11	Vynil Tubing	1	\$9.97
	12	Copper rod	1	\$25.49
	13	Smaller Hose Clamps	1	\$2.99
1	14	ABS Filament	1	\$21.99
	15	Rubber Whip Hose	1	\$8.62
0.47	16	Arduino	1	\$19.99
	17	Temperature sensor	1	\$9.99
	18	Speaker	1	\$11.99
	19	Teflon Tape	1	\$4.99
	20	Pipe Insulation	1	\$47.92
	21	Inside pipe (Tigerflex)	60	\$30.00
	22	Anti-reflective glass	1	\$60.00
			Real	\$531.66

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COMSOL Problem Setup

- L-VEL turbulence model
- Heat transfer simulated in steady-state
- Natural convection on exterior surfaces
- Pressure at the air inlet of the model tested at 0.1 psi and 1.0 psi
- Air enters the device at ambient temperature
- Heat generation of the camera is negligible

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Streamlines of the Velocity Field



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Surface Temperature of the Camera



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Cost of Operation

General assumptions for estimating the cost of operation:

- The price of electricity is \$0.10/kWh
- The set point of the compressor is 100 psi
- The compressor consumes 20 kW per 100 cfm of compressed air at 100 psi

Note: Solutions from the COMSOL simulation for the volumetric flow rate were used together with the cost per cubic meter of compressed air found from internet sources to get an estimate for the cost of operating the device.

20 RW	\$0.10	100 cfm	1 h	- \$0.012 por m ³
100 cfm	1 kWh	4.72 x 10 ⁻² m ³ /s	3600 \$	- <u>30.012 per m</u>



Cost of Operation per 24 hours vs Inlet Pressure

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Validation

Approximating camera heat generation with thermocouples to validate its heat contribution to the system is negligible





 $T_{\text{insulation}} = 28.5^{\circ}\text{C}$ T_{camera} = 33 °C Heat $Flux = \frac{Q}{r} =$ $k\Delta T$ thickness *Heat Flux* = 27 W/m^2 $Q = 27 W/m^2 x A_s \approx 0.0025 W$

5 mm

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Validation

Target	Metric	Simulated Results
Camera Temperature	0 ≤ T ≤ 45°C	14.1 ≤ T ≤ 34.8°C
Days of operation	71	Indefinite
Adequate Lighting	Yes	N/A
Inexpensive camera	< \$100	\$49.99
Lens Condensation	0 mL	0mL
Camera Heat Generation	Negligible	Negligible

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"It's not a problem it's an opportunity"





This is the end of the Presentation

Backup Slides

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